Thermodynamic analysis of the C-13-1 steam catapult for aircraft launching from an aircraft carrier

Análisis termodinámico de la catapulta de vapor C-13-1 para lanzamiento de aeronaves desde un portaviones

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Abstract

This manuscript presents a thermodynamic analysis of thermal energy storage regarding C-13-1 catapult used to launch aircraft from the USS Nimitz CVN-68. The results showed a steam injection coefficient of 4.4%. In this way, the simulated accumulator reduces steam thermal power supplied in batch mode from 530 MW required for launch to 22.9 MW, provided continuously by the generator throughout the whole cycle. The total thermal energy consumed was 1389 MJ. The forces exerted during take off an FA-18 Hornet indicated 85.9% action for the catapult and 14.1% for the turbofans, with 1.9% corresponding to friction forces on the total traction.

Keywords

Flash steam, steam accumulator, thermal energy storage, steam generator, batch process.

Resumen

En este artículo se realizó el análisis termodinámico de la acumulación de vapor para almacenamiento de energía térmica. El sistema analizado fue la catapulta C-13-1 utilizada para el lanzamiento de aeronaves desde el USS Nimitz CVN-68. Los resultados mostraron un coeficiente de inyección de vapor de 4,4%. De esta forma, el acumulador simulado reduce la potencia térmica del vapor suministrado a la catapulta desde los 530 MW necesarios en forma batch por lanzamiento a 22,9 MW aportados por el generador de forma continua en todo el ciclo; la energía térmica consumida son 1.389 MJ. Las fuerzas ejercidas durante el despegue de un FA-18 Hornet indicaron el 85,9% de acción para la catapulta y el 14,1% para los turbofan, de los cuales el 1,9% correpondían a las fuerzas de fricción sobre la tracción total.

Palabras clave

Vapor flash, acumulador de vapor, almacenamiento de energía térmica, generador de vapor, proceso por lotes.

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USS Nimitz CVN-68 aircraft carrier (Atalayar, 2021).

1. Introduction

Steam accumulators are used as thermal energy storage to balance steam fluctuations between supply and consumption. These systems considerably improve the operating conditions and quality of the steam supplied, saving thermal energy. However, steam accumulators are difficult to implement due to the high initial investment costs required, as they depend mainly on the volume and pressure needed (Sung et al., 2016). Batch processes are a typical example of this type of situation, where a huge amount of steam is required for short periods. Steam generators cannot react instantaneously to steep modifications in demand (Biglia et al., 2017).

A steam accumulator saves energy, reduces pressure fluctuations and the aging of pressurised lines and vessels in steam generator. There are two thermodynamic models for calculating steam accumulator units: equilibrium and non-equilibrium models. The first one is based on equilibrium equations associated with the global mass and energy balance of water and steam content in the tank. The second one uses mass and energy equations for each phase and establishes non-equilibrium evaporation and condensation rates. The dynamic changes in steam consumption cause pressure changes in pipes and pressurised vessels, originating dynamic thermomechanical loads that endanger equipment integrity. These adverse effects are reduced when steam accumulators are

installed (Stevanovic et al., 2015).

The energy efficiency of a steam generator decreases rapidly with increasing the fluctuation frequency of the consumer steam load. This efficiency is increased by 3-11% when a steam accumulator is installed (Jiacong, 2000). The charging process constitutes the supply of saturated or superheated steam from the steam generator to the steam accumulator. The unloading causes a pressure reduction of the saturated water while releasing flash steam (Yang et al., 2017). An example includes the use of steam for applications in which work is carried out for a short period, such as steam catapults for launching aircraft or in power plants during peak loads (Stevanovic et al., 2012). Beside steam accumulators, other technologies commercially available for thermal energy storage include the two tank of molten salts with application in concentrated solar power plants (González-Roubaud et al., 2017).

Direct storage of saturated or superheated steam in pressure vessels is not economical due to its low energy density. Water is the preferred storage medium below 100 °C because of the high specific heat value, great availability, environmental safety and low cost (Stark et al., 2017). The steam accumulator is an essential part of the steam catapult of an aircraft carrier, as it provides the necessary amount and pressure of saturated steam in an extremely short time, ensuring that the aircraft acquires the take off speed within the short deck length. The fact that the steam catapult system propels an aircraft of more than 20 t at take off speeds of more than 200 km/h in about 2 s, gives an idea of the strong imbalance the accumulator is submitted (Sun et al., 2015).

Hydraulic catapults gave way to steam catapults in the 1950s, while in the early 2000s, an alternative technology for launching aircraft was developed, called the electromagnetic catapult, which is powered by linear induction motors working on the same basic principles as electric induction motors. The United States Navy developed the electromagnetic aircraft launch system (EMALS) which was installed on the USS Gerald R. Ford (CVN-78) aircraft carrier, the first of a new generation (Parwate et al., 2017). However, this new technology has a high cost compared with the steam catapult system.

Objetive

This manuscript aimed to analyze thermal energy storage by a steam accumulator using high pressure saturated water to supply steam to a C-13-1 catapult for launching aircraft from the CVN-68 aircraft carrier deck. The kinematics and dynamics of the entire aircraft launching system and the mass and energy balance of the steam system at the operating point were studied. This study allows understanding the interaction between forces developed by the catapult and turbofan for different aircraft take off scenarios.

2. Materials and methods

2.1. Description of the scenario

The C-13-1 steam catapult currently installed on the CVN-68 aircraft carrier (first in the series) consist of two rows of slotted cylinders inserted into a channel 1.07 m deep and 1.42 m wide located directly below the fly deck (Zhou & Huang, 2020). The main characteristics of the CVN-68

are: displacement 101000 t, length 328.3 m, width 76.8 m, draught 11.3 m, 2 nuclear reactors A4W (aircraft, fourth generation, Westinghouse) of 104 MW, 4 steam turbines and 4 propellers, cruise speed 56 km/h, crew 3200 people, 82 aircraft, 4 lifts and 4 steam catapult C-13-1 (F16, 2020). Table 1 presents the main data for three steam catapults, along with the aircraft carriers,

aircraft take off speed, aircraft maximum operational weight, among others.

MTOW, take off speed and cycle time constitute the "operating point" of the C-13-1 catapult for the calculation. All other parameters of the C-13-1 steam catapult and main characteristics of the FA-18 Hornet were based on data shown in Table 2.

Parameters	C-13-0	C-13-1	C-13-2	References
Aircraft carrier	Kitti Hawk, CV-63 Constellation, CV-64 Enterprise, CV-65 America, CV-66 Kennedy, CV-67(3)	Kennedy, CV-67(1) Nimitz, CVN-68 Eisenhower, CVN-69 Vinson, CVN-70 Roosvelt, CVN-71	Lincoln, CVN-72 Washington, CVN-73 Stennis, CVN-74 Truman, CVN-75 Reagan, CVN-76	F16, 2020
Cylinder stroke length (m)	75.9	94.2	94.2	Global Security, 2020
Length of take off runway (m)	80.7	99	99	Global Security, 2020; F16, 2020
Piston and shuttle mass (kg)	2880.4	2880.4	2880.4	
Number of cylinders (uts)	2	2	2	Global Security, 2020
Cylinder diameter (mm)	457.2	457.2	533.4	
Take off speed (km/h)	237.1	259.3	259.3	Global Security, 2020; F16, 2020
MTOW (kg)	33566	36288	36288	Global Security, 2020; Naval education and training com- mand, 1974

Table 1. Main data of three classes of catapults installed on USS aircraft carriers.

Parameters	Values	References						
C-13-1- Catapult								
Complete time in each launch (s)	60	Shi et al., 2008						
Steam pressure in the secondary circuit (kPa)	6900	Elward, 2010						
Friction coefficient between piston and cylinder (w.u)	0.081	The engineering Toolbox1, 2020						
Radius of the shuttle wheel (mm)	150	Quora, 2020						
Rolling resistance coefficient between shuttle wheels and steel rail (mm)	0.5	The engineering Toolbox2, 2020						
Total length of braking (m)	1.5	Global security, 2016						
Length of water brakes (m)	2.7	F16, 2020						
FA-18 Hornet								
Unladen weight (kg)	11325							
MTOW unloaded (kg)	16100							
MTOW loaded (kg)	23590							
Number of turbofan (uts/each)	2							
Thrust of the turbofan (N/each)	52200	Swiss Armed Force, 2020						
Thrust of the turbofan with postcombustion (N/each)	78399							
Lenght (m)	17.1							
Width (m)	11.7							
Height (m)	4.6							
Wing area (m ²)	38	FA-18 Hornet, 2013						
Angle of the holdback bar (°)	26	Quere 0000						
Angle of the launch bar (°)	40	Quora, 2020						
Dynamic coefficient of the tyre inflation pressure (w.u)	0.009	Lénoz 0012						
Static coefficient of the tyre inflation pressure (w.u)	0.002	Lopez, 2013						
Aerodynamic drag coefficient (w.u)	0.02	Muñoz Navarro, 2020						

Table 2. Calculation parameters of the C-13-1 catapult and FA-18 Hornet.



Figure 1. General operating diagram of the C-13-1 steam catapult. SG, steam generator; T, steam turbine; C, condenser; P, propeller and G, electric generator.

2.2. Description of the operation of the C-13-1 steam catapult

The operating system mainly consists of: (1) steam supply from the secondary circuit of nuclear reactor, (2) inlet valve, (3) steam accumulator, (4) launch valve, (5) cylinders and pistons, (6) water brakes, (7) cylinder preheating system, (8) firefighting system, (9) drainage line, (10) retraction system, (11) cylinders lubrication system, (12) deck control cabin and (13) jet blast deflector (JBD).

The pressurized saturated water primary circuit of the two A4W reactors, pressurized water reactors (PWR) and UO_2 fuel, transfers the thermal energy to the steam generator (secondary circuit). Water from the condenser in this circuit changes to saturated steam. The system operates as a Rankine cycle. The four propellers, electrical generators, auxiliary services and four catapults of the aircraft carrier are driven by the energy of steam.

The accumulator contains saturated water at the same pressure and temperature as the inlet steam. An instantaneous pressure change occurs when the outlet accumulator steam valve opens (launch valve). Flash steam is generated at lower pressure and temperature, and it is sent to the two cylinders of the catapult. Steam expansion in the cylinders activates the two pistons transmitting movement to a mechanism called the launch bar, to which the aircraft is attached by the front landing gear. Thus, take off speed is achieved in a short time, combining the catapult traction force and turbofan full load thrust. operating without post-combustion.

The cylinder preheating system allows valves, pipes, cylinders, pistons and all elements between the accumulator and the catapult to be slowly preheated to increase



Figure 2. Section of the catapult launch system. a) Cross (F16, 2020). b) Longitudinal (Navy BMR, 2020).

temperature to values close to operating conditions (Global security, 2020). Water brakes slow piston speed to a complete stop at the end of the cylinder stroke, dissipating their kinetic energy against the water and the deck structure. The retraction system allows pistons to return to their initial position when the launch is finished and the steam used is sent to the drainage steam line by opening the exhaust valve. The lubrication system reduces friction between cylinders and pistons. The JBD protects the operators and other aircrafts against the turbofan exhaust gases and it forms a 50° angle with the horizontal in order to deflect it to the atmosphere (SBIR-STTR, 2019). Figure 1 shows the general operating diagram of the C-13-1 steam catapult.

Each catapult has two rows of launching cylinders mounted in parallel in the deck channel. The cylinder cover acts as a clamp that holds the slotted part of the cylinder in position to prevent radial sca-



Figure 3. Water brakes. a) Spear about before entering. b) Spear fully inserted.

ttering when applying steam pressure. Figure 2a shows the cross section and Figure 2b the longitudinal section of the catapult launch system.

Figure 3a represents when the piston approaches the water brake at take off speed. The brakes are always full of water before the impact. This is done by continuously injecting water under pressure into the jet ring, giving a vortex movement to the jet so that it moves helically along the walls of the brake towards the rear, leaving a space in the center with only air. When the water reaches the vane of the brake end cap, the vortex is broken and water flows through the central part of the brake in the opposite direction, displacing the air and flowing towards the water tank in an existing opening next to the choke ring, constituting a closed water circuit. Figure 3b shows the piston stopping moment with the spear fully inserted into the brake and water flowing out completely from the duct next to the choke ring into the water tank. Once the piston is retired, a pump introduces water again into the jet ring to keep the brake full of water in a continuous movement until the next release (Garstin, 2016).

A complete launch cycle consists of: (1) approach of the aircraft to the launch site, (2) lifting of JBD, (3) launch and holdback bar coupling, (4) setting of the turbofan power to the maximum without post-combustion, (5) launch, (6) braking and (7), retraction of pistons to the launch site, extraction of excess steam, filling of water brakes and lowering of JBD (Navy BMR, 2020).

2.3 Description of equations

Equation 1 allows calculating shuttle and aircraft acceleration without considering any friction. Equations 2 and 3 estimate the take off and catapult operating time, assuming as a simplification that acceleration remains constant from the end of the catapult (94.2 m) to the runway (99.0 m), although in this small section of the track, the aircraft is only operated by the two turbofans.

$$a_{cat} = \frac{\left(v_{final-cat}^2 - v_{inicial-cat}^2\right)}{2 \cdot L_{cat}} \quad (1)$$

$$t_{\text{take off}} = \sqrt{\frac{2 * L_{\text{track}}}{a_{\text{cat}}}}$$
 (2)

$$t_{cat} = \sqrt{\frac{2 \cdot L_{cat}}{a_{cat}}}$$
(3)

Equation 4 is used for estimating the energy imparted by the catapult to allow aircraft take off, while Equation 5 calculates the power associated.

$$E_{\text{cat-aircraft}} = \frac{1}{2} * (m_{\text{aircraft}} + m_{\text{piston}}) * v_{\text{final-cat}}^2 * 10^{-3} \text{ (4)}$$

$$P_{\text{cat-aircraft}} = \frac{E_{\text{cat-aircraft}}}{t_{\text{cat}}} \text{ (5)}$$

Equation 6 calculates the traction force required for the catapult and associated with the expansion of steam into the cylinders. Equation 7 calculates the speed reached by the airplane when driven by turbofans (k = 0.8), considering in both cases the friction associated with the launch system and the aircraft (Hernando-Díaz, 2018).

$$F_{cat} = 1.02 * \left(\frac{m_{piston} + m_{aircraft}}{2}\right) * \frac{v_{final-cat}^2}{L_{cat}} (6)$$

$$v_{\text{final-cat}} = \sqrt{2 * L_{cat} * (\frac{E_{turbofan}}{m_{aircraft}}) * k}$$
 (7)

Equation 8 expresses the negative braking acceleration of the pistons obtained after their complete stop in the water brakes. Equation 9 the braking time, which is the time needed for pistons to stop. Equation 10 the exerted force of the catapult shuttle. Equation 11 the braking energy of the pistons and Equation 12 the average braking power dissipated in the water brakes of cylinders.



Garstin (2016) considers that steam expansion in the cylinders is not adiabatic for the whole piston stroke. However, Shi et al. (2018) assume adiabatic expansion. When assuming the ideal pressure value, the pressure starts from zero, rises almost linearly until it reaches its nominal value at the point corresponding to 10% of the cylinder stroke position and then remains constant for the remaining 90% of the stroke. On the contrary, when assuming real pressure values, only when reaching 43% of the stroke position and up to the end, the actual expansion resembles an adiabatic process (Garstin, 2016). For simplification, steam was assumed to expand ideally from the beginning to the end, in an isobaric way inside the cylinders. The thermodynamic model was the equilibrium model. Figure 4 shows a generic diagram of a steam accumulator (Spiraxsarco, 2017).

Equation 13 estimates the accumulator absolute steam pressure required for aircraft launching, in accordance with C-13-1 catapult design parameters, considering that atmospheric pressure opposes movement. Although the launch steam pressure can vary between certain values (3061-3537 kPa, Roosvelt aircraft carrier, CVN-71) (EPA, 1999), in this work it was considered constant and estimated as a function of the operating point.

$$p_{cat} = \frac{4 * F_{cat} * 10^{-3}}{N_{cy} * \pi * \Phi^2} + p_{atm} \quad (13)$$

The steam mass flow required for a launch is calculated using Equation 14.



Drainage valve



$$M_{\text{steam-batch}} = \frac{N_{\text{cy}} * \pi * \Phi^{2*} L_{\text{cat}} * \rho_{\text{steam}} * 3,600}{4 * t_{\text{cat}}} (14)$$

Equation 15 estimates mean steam consumption. This value is coincident with the steam mass flow from the steam generator in a continuous manner. Equation 16 is used to obtain the steam mass flow based on the steam injection coefficient.

$$M_{\text{steam-mean}} = \frac{M_{\text{steam-batch}^{\text{t}} \text{tcat}}}{t_{\text{cycle}}} (15)$$

$$k_i = \frac{t_{cat}}{t_{cycle}} * 100$$
 (16)

The thermodynamic model adopted for evaluating the steam accumulator was based on equilibrium. The steam mass flow delivered to the catapult is supplied by the steam generator operating continuously. The steam accumulator only works during the steam injection phase (batch conditions). Equation 17 represents the mass flow of flash steam produced by the accumulator as the difference between the steam supplied in each batch and the mean steam value (as mass flow).

$$M_{acu} = M_{steam-batch} - M_{steam-mean}$$
 (17)

Equation 18 represents the amount of saturated water stored in the steam accumulator needed to supply steam to the catapult. Equation 19 represents the amount of flash steam sent to the catapult and expressed as percentage.

$$m_{\text{sat-acu}} = \frac{M_{\text{acu}} * t_{\text{cat}}}{\% \text{flash steam}}$$
(10)
%flash steam = $\frac{(h_{\text{P1}} - h_{\text{P2}}) * 100}{(10)}$

(10)

%flash steam =
$$\frac{(1112)}{h_{change-state-P2}}$$
 (19)

Equation 20 is used to calculate the volume of the accumulator considering a filling factor of 0.9 (Wenqiang et al., 2017). Dimensions of the cylindrical tank were calculated with a length/diameter ratio of 5 (Stevanovic et al., 2012).

$$V_{acu} = \frac{m_{sat-acu}}{\rho_1^* f_{filling}}$$
(20)

Equation 21 gives the degree of steam released from the accumulator. Equation 22 the area of the cylinder cap obtained after applying the filling factor. Equation 23 the water free surface area and Equation 24 the maximum value of steam release degree (Spiraxsarco, 2017).

$$G_{\text{release-steam}} = \frac{M_{\text{acu}}}{S_{\text{free water surface}}}$$
 (21)

$$A_{cap} = \frac{R_{acu}^2}{2} * (\varphi - sen\varphi) = (1 - f_{filling}) * \pi * R_{acu}^2$$
 (22)

$$S_{\text{freewatersurface}} = L_{acu} * 2 * R * sen(\frac{\varphi}{2})$$
 (23)

Figure 5a shows in a simplified way, forces considered during the launch of the FA-18 Hornet aircraft from a C-13-1 catapult. It was assumed that all forces act on point O (without con-



Figure 5. Diagram of shuttle and front landing gear with its launch and holdback bars. a) Acting forces. b) Force diagram.

sidering moments) where the shuttle is attached to the launch bar. Figure 5b shows the force diagram, acting on the shuttle, where point O is located close to the deck.

Equation 25 represents the "launch equation" for the FA-18 Hornet. Equation 26 is used to calculate the friction force between pistons and cylinders. Equation 27 and 28 are used to calculate the friction force and coefficient associated with the shuttle wheels, whereas Equation 29 and 30 for estimating friction for the aircraft wheels (López, 2013).

$$\frac{F_{cat} - F_{FP-C} - F_{FR-S} - F_{FR-A} - F_{FA-A}}{(m_{sizer} + m_{rister}) * a_{takeoff}}$$
(25)

$$F_{FP-C} = \mu_{FP-C} * g * m_{piston}$$
(26)
$$F_{FR-S} = \mu_{FR-S} * F_{Cat} * tg\alpha$$
(27)

 $F_{FR-S} = \mu_{FR-S} * F_{Cat} * tg\alpha \qquad (27)$

 $\mu_{\text{FR-S}} = \frac{r}{r} \tag{28}$

 $F_{FR-A} = \mu_{FR-A} * g * m_{aircraft}$ (29)

$$\mu_{\text{FR-A}} = f_0 + f_s * 24.5 * \left(\frac{v_{\text{aircraft}}}{100}\right)^{2,3}$$
(30)

Equation 31 calculates the aircraft aerodynamic friction force, using in this case the aircraft speed in m/s as units.

$$F_{FA-A} = \frac{1}{2} * \rho_{air} * (v_{aircraft})^{2*} s_{wing} * c_D \quad (31)$$

3. Results and discussion

3.1. Shuttle analysis

Table 3 shows results from the analysis of elements coupled to the shuttle: catapult, aircraft

take off and piston braking. The catapult nominal launch capacity (36288 kg) was divided into the aircraft mass to be launched (33408 kg) and shuttle-piston mass (2880 kg).

The aircraft acceleration from the point of shuttle release (94.2 m)

and the runway end (99.0 m) was assumed constant, even though only the turbofan thrust acts on this section (3.8 m). The difference in speed and final time between catapult and aircraft take off is only 2.4%; this fully justifies the assumption for the catapult and

Parameters	Values					
Catapult						
Mean acceleration (m/s ²)	27.5					
Mean acceleration (g)	2.8					
Final speed (m/s)	72					
Launch time (s)	2.62					
Traction force without friction (kN)	999					
Traction force with friction (kN)	1019					
Friction force (kN)	20					
Energy with friction (MJ)	96					
Energy without friction (MJ)	94.1					
Friction energy (MJ)	1.9					
Power with friction (MW)	37.4					
Power without friction (MW)	35.9					
Friction power (MW)	1.5					
Ai	rcraft					
Mean acceleration (m/s ²)	27.5					
Launch time (s)	2.62					
Final speed (m/s)	73.8					
Br	akes					
Braking acceleration (m/s ²)	1728					
Braking acceleration (g)	176					
Braking time (s)	0.042					
Braking energy (MJ)	7.5					
Braking power (MW)	178					
Reaction force in structure (kN)	5000					
Final water temperature (°C)	142					

Table 3. Results from the shuttle analysis.

take off parameter as equivalent at the operating point. The shuttle transmits a horizontal traction force to the aircraft of 1019 kN (when friction is considered). This force is supplied in 2.62 s by steam from the accumulator. A 2% loss was assumed in Equation 6. The negative acceleration exerted on pistons by water brakes has a value of 1728 m/s2, generating a reaction on the deck fuselage equivalent to 5000 kN (510 t). This value corresponds to 0.5% of the aircraft carrier mass and raises the water temperature up to 142 °C.

3.2. Steam system analysis

Figure 6a shows represents the flow of steam supplied by the accumulator to the catapult and Figure 6b, the steam thermal power. Figure 6a also represents the flow of steam supplied by the boiler for each operating cycle as a function of the launch time at the operating point. The catapult delivers a mass flow of 680923 kg/h of steam in just 2.62 s in a cyclic way of 60 s duration. The mean value of steam supplied to the catapult was 29734 kg/h. This flow is produced by the steam generator, resulting in an injection coefficient of 4.4%. This means that during 4.4% of the launch cycle time, steam is injected into the catapult, while in the remaining time, energy is continuously stored in the accumulator. The accumulator supplies 651189 kg/h (95.6%) of steam which, added to the 29734 kg/h (4.4%) from the steam generator, constitutes the total value of 680923 kg/h (100%) required by each catapult cycle.

Results from the generator, accumulator and catapult are shown in Table 4. A horizontal configuration was assumed instead of a vertical one for obvious reasons of space in the aircraft carrier and because of the larger evaporation area provided by this configuration which depends on saturated free surface area in the tank.

The steam accumulator, when filled to 90% of its total capacity, has a free evaporation area of 68.3 m^2 (3.15 m × 21.68 m) and a circular cap angle of 93.2°, re-



Figure 6. Diagram of steam supply to the steam accumulator as a function of launch cycle time. a) Mass flow. b) Thermal power.

sulting in a steam release rate of 9,532 kg/h*m². This value is lower than the maximum release rate of 15,180 kg/h*m² (Spiraxsarco, 2017). The ratio between high and low manometric pressures was 2.2:1. Table 5 presents the mass and energy balance of the steam accumulator for the complete cycle at the operating point (injection and non-injection). It is necessary to supply 0.3 MW additional to the energy provided by the accumulator to compensate the energy balance, due to the difference between the inlet specific enthalpy and outlet. This power of 0.3 MW represents just 1.3% of the total input thermal power, and therefore this amount was neglected.

The steam accumulator, when

Parameters	Values						
Generator							
Steam mass flow (kg/h)	29734						
Steam pressure (kPa)	6900						
Water saturation temperature (°C)	284.9						
Steam thermal power (MW)	22.9						
Acumulator							
Outlet steam pressure (kPa)	3205						
Manometric pressure ratio (w.u)	2.2						
Outlet steam temperature (°C)	237.6						
Outlet steam mass flow (kg/h)	651189						
Thermal power outlet steam (MW)	507.1						
Steam mass storage required (kg/cycle)	28435						
Proportion of flash steam (%)	13.13						
Amount of saturated water required (kg)	213790						
Tank filling coefficient (w.u)	0.90						
Water mass in full tank (kg)	237544						
Density (saturated water at 237.6 °C (kg/m ³)	741.5						
Minimum tank volume (m³)	320						
Diameter (Length/Diameter = 5) (m)	4.34						
Accumulator lenght (m)	21.68						
Energy density, volumetric storage (kWh/°C*m³)	0.9						
Accumulator energy density (kWh/m³)	260.3						
Launch steam mass/Accumulator water mass (kg/kg)	0.002						
Cap cylinder area (m²)	1.48						
Evaporation free surface area (m ²)	68.3						
Accumulator cap central angle (rad)	1.63						
Steam release degree (kg/h*m ²)	9532						
Steam release maximum degree (kg/h*m ²)	15180						
Steam injection coefficient (%)	4.4						
Catapult (with friction)							
Steam mass flow (kg/h)	680923						
Launch time (s)	2.62						
Preparation time for launch (s)	57.38						
Launch cycle time (s)	60						
Steam mass (kg/launch)	496						
Steam energy (MJ/launch)	1389						
Steam power (MW/launch)	530						

Table 4. Results from the steam system analysis.

filled to 90% of its total capacity, has a free evaporation area of 68.3 m^2 (3.15 m × 21.68 m) and a circular cap angle of 93.2°, resulting in a steam release rate of 9532 kg/h*m². This value is lower than the maximum release rate of 15180 kg/h*m² (Spiraxsarco, 2017). The ratio between high and low manometric pressures was 2.2:1. Table 5 presents the mass and energy balance of the

steam accumulator for the complete cycle at the operating point (injection and non-injection). It is necessary to supply 0.3 MW additional to the energy provided by the accumulator to compensate the energy balance, due to the difference between the inlet specific enthalpy and outlet. This power of 0.3 MW represents just 1.3% of the total input thermal power, and therefore this amount was neglected.

Figure 7a shows the temperature-pressure diagram of water-steam for the process in the accumulator and its comparison with the steam generator or boiler (operating without an accumulator) to achieve the same aim. The boiler is incapable of supplying the high steam flow needed in such a short time. Figure 7b compares the generator steam flow (4.4%) and that

Parameters	Inlet steam (Generator)	Saturated water (Accumulator)	Outlet steam (Catapult)			
	Continuous	Variable level	Batch	*Continuous		
Steam mass flow (kg/h)	29734		680923	29734		
Thermal power (MW)	22.9		530.2	23.2		
Temperature (°C)	284.9		23	7.6		
Pressure (kPa)	6900		32	05		
Thermal power losses (MW)		0.3				
Maximum mass of water (kg)		213790				
Maximum volumen of water (m3)		288.3				
Minimum mass of water (kg)		213294				
Minimum volume of water (m ³)		287.7				
*Medium cycle						

Table 5. Mass and energy balance of the steam accumulator.

of the accumulator (95.6%) during each launch. Figure 7c and Figure 7d illustrate the process of batch and non-batch steam injection of the steam accumulator, while the steam supply from the steam generator is constant throughout the cycle.

3.3. Analysis of the nominal traction capacity of the C-13-1 steam catapult

The effect of varying pressure and take off speed are analyzed. When Equation 6 is applied, a set of curves with a hyperbolic shape for each value of the traction force and steam pressure are obtained. Figures 8a and 8b show the evolution of the aircraft mass that can be launched as a function of the take off speed for each value of the catapult traction force and steam pressure in the accumulator. The black line in both figures indicates the nominal operating curve of the C-13-1 catapult and the red dotting arrow, the operating point. These curves were obtained by sealing off the thrust of the aircraft turbofans, only with the action of catapult.

3.4. Analysis of an FA-18 Hornet take off The take off of an FA-18 Hornet from an aircraft carrier or an airport with its MTOW has been analysed in a simplified way, assuming the effect of friction forces and also neglecting them. Equation 32 corresponds to the equation for the aircraft launch using





Figure 7. Steam accumulator. a) Diagram of the water-steam circuit. b) Comparison with the steam mass flow of the catapult. c) and d) Processes batch and no-batch in the steam accumulator.



Figure 8. Sensitivity analysis for the mass of the piston and aircraft to be launched by the catapult as a function of the take off speed. a) Traction force on the shutt-le. b) Steam pressure during launch.

only steam catapult and Equation 1 33 for turbofan, considering in t

both cases friction forces.

+0.4535*v2aircaft)*10-3

+0.4535*v2aircraft)*10-3

(33) These equations clearly show that force (and therefore, acceleration) is variable and a function of the launch speed. However, the variation of acceleration is small, so the mean acceleration has always been considered. Table 6 shows each of the components of friction forces during the take off in accordance with four scenarios of the launch: (a) catapult, (b) catapult and turbofan, (c) turbofan in catapult track and (d) turbofan in airport track.

For Scenario (b), the following percentage of forces acting on the aircraft results: steam catapult 85.9% and turbofan 14.1%, approximately a 6/1 ratio. The results show the low incidence of friction, rolling and aerodynamic forces on the total force required for take off compared to that required to accelerate the aircraft-piston assembly. The results evidence the small incidence of friction, rolling and aerodynamic forces on the total force needed for launching compared with that required to accelerate the aircraft-piston assembly. The force required by the FA-18 Hornet for take off is 72.1% of the nominal catapult force when friction is neglected

and 72.8% if considered. In this way, 98.1% of the catapult force is used to accelerate the aircraft-piston assembly and the remaining 1.9% is used to counteract the friction force that opposes the launch movement. Results are identical in the case of accelerating the aircraft with the catapult and turbofan providing the force. In the case of turbofan thrust and catapult track, the resulting friction force is 2.8%. When the take off takes place on the airport track, the friction force is raised to 9.1%, due to the increase of speed with respect to the previous case and with this value being much higher than that obtained for the launch using when the force provided by the catapult. The difference between the friction and frictionless force (or

Parameters	Scenario (a)		Scenario (b)		Scenario (c)		Scenario (d)	
	Departure	Take off						
Traction force of the catapult (kN)	734.8	724.1	630.4	637.7	0	0	0	0
Turbofan thrust (kN)	0	0	104.4	104.4	104.4	104.4	104.4	104.4
Friction force of the shuttle wheels (kN)	2.1	2.1	1.8	1.8	0	0	0	0
Friction force of piston-cylinder (kN)	2.3	2.3	2.3	2.3	0	0	0	0
Friction force of aircraft wheels (kN)	2.1	7.1	2.1	7.1	2.1	2.5	2.1	7.1
Aerodynamic friction force (kN)	0	2.4	0	2.4	0	2.4	0	2.4
Total friction force (kN)	6.5	13.9	6.2	13.6	2.1	2.9	2.1	9.5
Traction force applied without friction (kN)	728.3	728.2	624.1	624.1	0	0	0	0
Turbofan thrust without friction (kN)	0 0		104.4	104.4	95	95	95	95
Friction force expressed as percentage of total traction or thrust force (%)	0.9 1.9		0.9	1.9	2	2.8	2	9.1

Table 6. Breakdown of the friction forces during the FA-18 Hornet take off.

thrust) with zero speed is due to the term f_0 of Equation 30. The resistance force caused by the air on the pistons was disregarded. Although the high piston speed causes a loss of air pressure when circulating through the cylinders. However, the air density is just 0.7 kg/m³ at 237.6 °C, thus the force resulting from the air opposing to the piston movement within cylinders is small and it was neglected.

Figure 9a represents the resulting forces required for the FA-18 Hornet launching, considering the action of either the catapult, turbofan or both on the catapult track. Figure 9b considers only turbofan on the catapult track and Figure 9c shows the forces acting on the aircraft in the case of operating just the turbofan on the airport track, as a function of speed in the three cases. In these figures, the cases considering friction and disregarding it are also represented. In Figure 9a there is a slight decrease in the value of the force exerted upwards by the launch bar, resulting in a lower rolling force of 0.3 kN at take off, a negligible value, so the launch Equations for Scenarios (a) and (b) have been considered equal.

The results from the take off of the FA-18 Hornet using three different forms of calculus are compared in Table 7: (1) applying the equations of classical physics without friction, (2) considering friction and (3) applying equations (8) and (9) based on the reference (Hernando-Díaz, 2018) for the Scenarios (a), (b), (c) and (d) listed in this same table. The mass of the aircraft and launcher-pistons is considered in Scenario (c) and for Scenario (d) only the mass of the aircraft. Total energy and power include friction.

Analysing case (2) with friction, the acceleration achieved and the take off time used by using catapult or catapult-turbofan in Scenarios (a) and (b) was 27.5 m/s² and 2.62 s, while only using the turbofan in Scenarios (c) and (d), the results were 3.9 m/s² (-85.8%) and 6.9-16.3 s (163.3%-522.1%), resulting in an insufficient final speed of 27.2 m/s (-62.2%) for take off in Scenario (c) and an airport take off length of 589.1 m (+525.4%) for Scenario (d). Considering the performance at the rated capacity of the steam catapult and turbofan, a 39739 kg aircraft can be launched at 72 m/s, 68.5% higher mass than the FA-18 Hornet. Obviously, with this capacity, it is also possible to launch heavier aircraft such as the F-35C.



Figure 9. Variation of the resulting force with and without friction as a function of the launch speed. a) Catapult and catapult-turbofan on catapult track. b) Only turbofan on catapult track. c) Only turbofan on airport track.

Parameters	Scenario (a)		Scenario (b)			Scenario (c)			Scenario (d)			
	(1)	(2)	(3)	(1)	(2)	(3)	(1)	(2)	(3)	(1)	(2)	(3)
Final speed (m/s)	72		72			27.2		24.2	72			
Mean acceleration (m/s ²)	27.5		27.5		3.9		3.1	4.4		3.5		
Launch time (s)	2.62		2.62		6.9		7.8	16.3	;	20.3		
Total energy (MJ)	68.6	69.9	70	68.6	69.9	70	9.8	10	10.1	61.1	62.4	66.7
Total power (MW)	26.2	26.7		26.2	26.7		2.5	2.6		13.9	14.2	15.2
Length of take off (m)	94.2		94.2		94.2			58	1.1	750.6		

Table 7. Parameters obtained of the take off of the FA-18 Hornet according to three different calculation processes.

4. Conclusions

The use of a steam accumulator reduces by 95.6% the thermal power that a steam generator of a nuclear reactor should supply to the catapult at each launch. The operation of all existing elements between the steam generator and accumulator can be continuous, avoiding dangerous thermal stresses and increasing equipment efficiency. It is important to highlight the great capacity for storing thermal energy that the accumulator allows by using saturated water (0.9 kWh/°C*m³ or 260.3 kWh/m3) and supplied in a short time (2.62 s) for each cycle (60 s). The analysis of forces acting during the launch shows that the catapult provides most of the force needed, whereas that provided by turbofans represents a small percentage. Friction forces are small when compared to the force provided by the set of catapult-turbofan system. The steam catapult is presented here as a way to highlight the relevance of steam accumulators in saving energy. In addition, the use of nuclear aircraft carriers greatly favors the use of saturated steam for this application.

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LIST OF SYMBOLS

a _{braking} : piston braking acceleration (m/s ²)	mpiston-aircraft: mass of pistons and aircraft (kg)
A _{enp} : area of the cylindrical cap of the accumulator (m ²)	msat-acu: mass of saturated water in the accumulator
acat: shuttle and aircraft acceleration (m/s ²)	Msteam-batch: steam mass flow of aircraft launch (kg/h)
atake-off: take off acceleration (m/s ²)	Msteam-mean: mean steam mass flow (kg/h)
c _D : aerodynamic drag coefficient (w.u)	MTOW: maximum take off weight (kg)
Ebraking: braking energy of the pistons (kJ)	N _{cy} : number of cylinders (uts)
Ecat-aircraft: catapult-aircraft energy at take off (kJ)	pacu: accumulation pressure (kPa)
Eturbofan: turbofan energy (kJ)	p _{atm} : atmospheric pressure (kPa)
F _{cat} : traction force exerted by the catapult (kN)	Pbraking: braking power of pistons (kW)
FFA-A: aerodynamic friction force (N)	pcat: absolute steam pressure of the catapult (kPa)
FFP-C: friction force between pistons and cylinders (N)	Pcat-aircraft: catapult-aircraft power at take off (kW)
FFR-A: friction force of the aircraft wheels (N)	r: shuttle wheel radius (mm)
FFR-S: friction force of the shuttle wheels (N)	Racu: accumulator radius (m)
Fturbofan: thrust exerted by turbofans without post-combustion (kN)	Sfree water surface: free water surface (m ²)
f _{filling} : filling factor (w.u)	swing: wing area (m ²)
fs: dynamic tire pressure coefficient (w.u)	t _{braking} : piston braking time (s)
fo: static tire pressure coefficient (w.u)	t _{cat} : catapult operation time per launch (s)
g: acceleration of gravity (m/s ²)	t _{cycle} : launch cycle time (s)
Grelease-steam: degree of steam release (kg/h*m ²)	trake-off: take off time (s)
Grelease-steam-max: máximum degree of steam release (kg/h*m2)	vaircraft: aircraft speed (m/s)
hchange-state: specific enthalpy of vaporisation at low pressure (kJ/kg)	Vacu: total accumulator volume (m ³)
h_{P1} : specific enthalpy of saturated water at high accumulator pressure (kJ/kg)	$\mathbf{v}_{\text{final-cat}}$: final speed of the catapult (m/s)
h_{P2} : specific enthalpy of saturated water at low accumulator pressure (kJ/kg)	vinicial-cat: initial speed of the catapult (m/s)
k: reduction coefficient (w.u)	α : angle formed between shuttle bar and deck (°)
ki: steam injection coefficient (w.u)	•: piston diameter (m)
\mathbf{k}_1 : coefficient of rolling resistance between shuttle wheels and rails (mm)	ρ_{air} : air density at 25 °C (kg/m ³)
L _{acu} : accumulator length (m)	Pateam: flash steam density (kg/m ³)
L _{cat} : catapult length (m)	$_{\rho_1}$: air density at storage tank temperature (kg/m ³)
Ltrack: track length (m)	φ: central angle of the accumulator cap (radians)
M_{acu} : flash steam mass flow (kg/h)	μ_{FP-C} : coefficient of friction between pistons and cylinders (w.u)
Lbraking: braking distance (m)	μ_{FR-A} : coefficient of rolling friction of aircraft wheels (w.u)
maircraft: aircraft mass (kg)	$\mu_{FR\text{-}s:}$ coefficient of rolling friction of the shuttle wheels $(w.u)$
Mgen: generator steam mass flow (kg/h)	%flash steam: flash steam percentage (%)
m _{piston} : piston mass (kg)	